Computational Fluid Dynamics 2

Turbulence effects and Particle transport

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Computational Biomechanics

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System equations:

mass conservation

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \, \vec{u}) = 0$$

momentum conservation

$$\rho \frac{\partial \vec{u}}{\partial t} + (\rho \, \vec{u} \cdot \nabla) \, \vec{u} = \rho \, \vec{f} + \nabla \cdot \underline{\underline{\sigma}}$$

energy conservation

$$\rho \frac{\partial e}{\partial t} = \rho \ Q + \nabla \cdot (\kappa \ \nabla \ T) + \nabla \cdot (\underline{\underline{\sigma}} \ \underline{u}) - (\nabla \cdot \underline{\underline{\sigma}}) \ \underline{u}$$

equation of state (e.g. ideal gas equation)

Incompressible flow:

Incompressible fluid:

$$0 = \frac{\mathrm{d}\rho}{\mathrm{d}t}(x,t)$$
$$= \frac{\partial}{\partial t}\rho(x,t) + \nabla\rho(x,t) \cdot \vec{u}$$

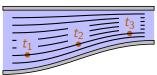
$$\frac{\partial \rho}{\partial t}(x,t) = \nabla \rho(x,t) = 0$$

Continuity equation:

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \, \vec{u}) = \underbrace{\frac{\partial \rho}{\partial t}}_{0, \text{fluid ass.}} + \underbrace{\nabla \rho \cdot \vec{u}}_{0, \text{fluid ass.}} + \rho \nabla \cdot \vec{u} = \rho \nabla \cdot \vec{u} = 0$$

0, flow ass.

• It follows: $\nabla \cdot \vec{u} = 0$ (divergency free velocity field)



Incompressible flow/fluid + isothermal assumption:

From T = const. with $\frac{d}{dt}\rho = 0$ follows:

- ① Pressure is given with $p \sim \rho$ (equation of state)
- ② Energy is a function of ρ and \vec{u} \Rightarrow the energy conservation contains no extra information

For a newtonian fluid we get the Navier-Stokes equations as

Navier-Stokes equations

$$\nabla \cdot \vec{u} = 0 \tag{1}$$

$$\frac{\partial \vec{u}}{\partial t} + (\vec{u} \cdot \nabla) \ \vec{u} = \vec{f} - \frac{1}{\rho} \nabla p + \nu \nabla \cdot \underline{\tau}$$
 (2)

Note: often, the kinematic viscosity $\nu := \frac{\mu}{\rho}$ is used

Dimensionless Navier-Stokes:

Navier-Stokes momentum equation

$$\frac{\partial \vec{u}}{\partial t} + (\vec{u} \cdot \nabla) \vec{u} = \vec{f} - \frac{1}{\rho} \nabla p + \frac{\mu}{\rho} \nabla \cdot \underline{\tau}$$

Define characteristic time T, length L and velocity U with $L = U \cdot T$:

$$\tau = \frac{t}{T} \qquad \vec{v} = \frac{\vec{u}}{U} \qquad \vec{\xi} = \frac{\vec{x}}{I}$$

Dimensionless representation of the momentum equation:

$$\frac{\partial \vec{v}}{\partial \tau} + (\vec{v} \cdot \nabla) \vec{v} = \frac{L}{U^2} \vec{f} - \frac{1}{\rho U^2} \nabla p + \frac{\mu}{\rho U L} \nabla \cdot \tilde{\underline{\tau}}$$

- dimensionless forcedensity $\vec{\kappa} := \frac{L}{II^2} \vec{f}$ (look for Froude number)
- pressure rescaling $\tilde{p} := \frac{p}{\sigma U^2}$ (NOTE: only for inc. fluid)

Diffusion term & Reynolds number:

$$rac{\partial ec{v}}{\partial au} + (ec{v} \cdot
abla) \ ec{v} = ec{\kappa} \ -
abla ilde{p} \ + rac{\mu}{
ho UL}
abla \cdot ilde{ ilde{ ilde{ ilde{ ilde{ ilde{ ilde{ ilde{v}}}}}} = ec{v} \cdot ec{ ilde{ ilde{v}}}$$

Definition of the Reynolds number:

$$Re := \frac{\textit{inertia forces}}{\textit{viscous forces}} = \frac{\rho \textit{UL}}{\mu}$$

- inertia force: $F_{in} = \frac{\rho L^3 \cdot U}{T}$ (momentum transfer)
- viscous force: $F_{vis} = \mu L^2 \cdot \frac{U}{I}$ ("velocity diffusion")

Dimensionless Navier-Stokes equations

$$\nabla \cdot \vec{\mathbf{v}} = 0 \tag{3}$$

$$\frac{\partial \vec{v}}{\partial \tau} + (\vec{v} \cdot \nabla) \ \vec{v} = \vec{\kappa} - \nabla \tilde{\rho} + \frac{1}{Re} \nabla \cdot \tilde{\underline{\tau}}$$
 (4)

Pressure equation:

$$\frac{\partial \vec{v}}{\partial \tau} + (\vec{v} \cdot \nabla) \vec{v} = \vec{\kappa} - \nabla \tilde{p} + \frac{1}{Re} \nabla \cdot \tilde{\underline{\tau}}$$

Divergency free velocity field implies

$$\nabla \cdot \left(\frac{\partial \vec{v}}{\partial \tau} + (\vec{v} \cdot \nabla) \ \vec{v} \right) = \nabla \cdot \left(\vec{\kappa} \ - \nabla \tilde{\rho} \ + \frac{1}{Re} \nabla \cdot \tilde{\underline{\tau}} \right)$$

with $\frac{\partial}{\partial x} \nabla \cdot \vec{v} = 0$, we get the Poissin-Pressure equation:

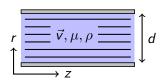
$$\Delta ilde{
ho} =
abla \cdot \left(ec{\kappa} \, - (ec{v} \cdot
abla) \, \, ec{v} \, + rac{1}{Re}
abla \cdot ilde{ au}
ight.$$

Turbulent flow:

- If Re << 1, the diffusion time scale is much smaller as the time scale for momentum transportation
 - velocity field perturbations smooth out quickly
 - velocity field <u>tends</u> to be laminar
- If Re >> 1, momentum transportation is the main effect for the fluid flow description
 - velocity field perturbations increase quickly
 - velocity field tends to be turbulent

Example: (flow in pipe)

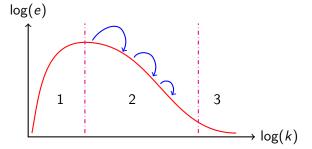
- Reynolds number: $Re = \frac{\rho d v_z}{\mu}$
- Observation: Julius Rotta (at 1950) $Re_{krit} \approx 2300$



Energy cascade:

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- energy injection range (small viscous effects)
- inertial subrange
- dissipation range (large viscous effects)



visualization after the model of Lewis Fry Richardson e := energy, k := wave number

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Kolmogorov scales:

The smallest scales that influences the turbulent flow by dissipation effects.

Note:

To retain energy conservation at the numerical domain, one have to resolve also the dissipative scales in the Navier-Stokes equation!

The scales are given as: (ϵ is the average dissipation rate)

$$\mathit{length}: \ \eta = \left(\frac{\mu^3}{\epsilon \, \rho^3}\right)^{\frac{1}{4}} \quad \mathit{vel}: \ u_\eta = \left(\frac{\mu}{\rho} \, \epsilon\right)^{\frac{1}{4}} \quad \mathit{time}: \ \tau_\eta = \left(\frac{\mu}{\rho \, \epsilon}\right)^{\frac{1}{2}}$$

with

$$extit{Re}_{\eta} = rac{\eta \ extit{u}_{\eta} \ \mu}{
ho} = 1$$

Resolution problem:

Approximation of the dissipation rate (from large scales):

$$\epsilon \sim rac{ ext{kinetic energy}}{ ext{time}} \sim rac{U^2}{T} = rac{U^3}{L}$$

Therefore we get the relation:

$$\frac{L}{\eta} = L \cdot \left(\frac{\mu^3}{\epsilon \rho^3}\right)^{-\frac{1}{4}} \sim L \cdot \left(\frac{U^3 \rho^3}{L \mu^3}\right)^{\frac{1}{4}} = Re^{\frac{3}{4}}$$

Example: $(L \approx 10^3 \mathrm{m}$, $v \approx 1 \frac{\mathrm{m}}{\mathrm{s}}$, $\rho \approx 1.3 \frac{\mathrm{kg}}{\mathrm{m}^3}$, $\mu \approx 17.1 \ \mu \mathrm{Pa \cdot s})$

$$Re \approx 7.5 \cdot 10^9$$
 $\eta \approx 4 \cdot 10^{-5} \,\mathrm{m}$

Resolution problem:

Approximation of the dissipation rate (from large scales):

$$\epsilon \sim rac{ ext{kinetic energy}}{ ext{time}} \sim rac{U^2}{T} = rac{U^3}{L}$$

Therefore we get the relation:

$$\frac{L}{\eta} = L \cdot \left(\frac{\mu^3}{\epsilon \, \rho^3}\right)^{-\frac{1}{4}} \sim L \cdot \left(\frac{U^3 \, \rho^3}{L \, \mu^3}\right)^{\frac{1}{4}} = Re^{\frac{3}{4}}$$

Example:
$$(L\approx 10^{-3} {
m m}$$
 , $v\approx 0.1~{
m m\over s}$, $\rho\approx 1060~{
m kg\over m^3}$, $\mu\approx 3~{
m mPa\cdot s})$
$$Re\approx 35$$

$$\eta\approx 7\cdot 10^{-5}\,{
m m}$$

Simulation approaches:

Direct numerical simulation (DNS):

Assumption that the flow inside of a volume element is purely laminar and no dissipation effect occurs. (Note: If this is not true. the energy conservation results in a different flow field.)

Eddy dissipation modelling on small scales:

- Reynolds-Averaged Navier Stokes (RANS)
- Large-Eddy Simulation

$$v = \langle v \rangle + v'$$
 and $p = \langle p \rangle + p'$

with the mean value $\langle \cdot \rangle$ of \cdot and the fluctuating part \cdot' .

RANS:

- Special cases: temporal or spatial averaging
- In general: $\langle f(\vec{x},t) \rangle = \lim_{N \to \infty} \sum_{n=1}^{N} f(\vec{x},t)$
- Fluctuating part: $\langle f' \rangle = 0$

Reynolds equations:

$$\begin{array}{c} \nabla \cdot \left\langle \vec{v} \right\rangle = 0 \\ \\ \frac{\partial \left\langle \vec{v} \right\rangle}{\partial t} + \left(\left\langle \vec{v} \right\rangle \cdot \nabla \right) \, \left\langle \vec{v} \right\rangle = \vec{f} \, - \nabla \left\langle p \right\rangle \, + \frac{1}{Re} \nabla \cdot \left\langle \underline{\tilde{\tau}} \right\rangle - \underbrace{\left\langle \left(\vec{v}' \cdot \nabla \right) \, \vec{v}' \right\rangle}_{\textit{correlation property}} \end{array}$$

$$\nabla \cdot \left\langle \vec{v}' \vec{v}' \right\rangle = \nabla \cdot \left(\begin{array}{cc} \left\langle v_x' v_x' \right\rangle & \left\langle v_x' v_y' \right\rangle & \left\langle v_x' v_z' \right\rangle \\ \left\langle v_y' v_x' \right\rangle & \left\langle v_y' v_y' \right\rangle & \left\langle v_y' v_z' \right\rangle \\ \left\langle v_z' v_x' \right\rangle & \left\langle v_z' v_y' \right\rangle & \left\langle v_z' v_z' \right\rangle \end{array} \right)$$

RANS models:

- Zero equation models $\nu_T = \xi^2 |\partial_\perp \langle v \rangle|$ (mixing length ξ)
- One equation models (example: Spalart and Allmaras)

$$\frac{\partial \nu_T}{\partial t} + \langle \vec{v} \rangle \nabla \nu_T = \nabla \left(\frac{\nu_T}{\sigma_T} \nabla \nu_T \right) + S_{\nu}$$

- Two equation models $(k \epsilon, k \omega, SST)$
 - $k = \frac{1}{2} \operatorname{tr} \langle \vec{v}' \vec{v}' \rangle$ (mean of the fluctuating kinetic energy)
 - dissipation rate ϵ
 - \bullet eddy frequency ω
 - **1** $k \epsilon$: good on free flow fields with no walls

 - SST brings the advantage of booth together

Large-Eddy simulations (LES):

spatial averaging method

$$\langle \vec{v}(\vec{x},t) \rangle := \int_{V} \vec{v}(\vec{x}',t) \cdot G(\vec{x},\vec{x}',\Delta) \ dV'$$

with

step-function

$$G := egin{cases} rac{1}{\Delta^3}, & \textit{if } |\vec{x} - \vec{x}'| < \Delta/2 \\ 0, & \textit{else} \end{cases}$$

gauss-filter

$$G := \mathcal{A}(\Delta) \; \exp\left\{rac{-eta \, |ec{x} - ec{x}'|}{\Delta^2}
ight\}$$

Large-Eddy simulations (LES):

LES equation:

$$\nabla \cdot \langle \vec{v} \rangle = 0$$

$$\frac{\partial \langle \vec{v} \rangle}{\partial t} + (\langle \vec{v} \rangle \cdot \nabla) \langle \vec{v} \rangle = \vec{f} - \nabla \langle p \rangle + \frac{1}{Re} \nabla \cdot \langle \tilde{\underline{\tau}} \rangle - \nabla \cdot \tau^{S}$$

with $\tau^S := \langle \vec{v}\vec{v} \rangle - \langle \vec{v} \rangle \langle \vec{v} \rangle$. Detailed look:

$$\tau^{\mathcal{S}} = \underbrace{\left\langle \left\langle \vec{v} \right\rangle \left\langle \vec{v} \right\rangle \right\rangle - \left\langle \vec{v} \right\rangle \left\langle \vec{v} \right\rangle}_{L} + \underbrace{\left\langle \left\langle \vec{v} \right\rangle \vec{v}' \right\rangle - \left\langle \vec{v}' \left\langle \vec{v} \right\rangle \right\rangle}_{C} + \underbrace{\left\langle \vec{v}' \vec{v}' \right\rangle}_{\tau^{SR}}$$

- Leonard-strain: creation of small eddys through large eddys
- Cross-stress: interaction of the different scales
- Subgrid-scale Reynolds stress tensor